



Crankcase. Pattern-making. Dowler.  
 Cylinder Head " " Mr Foster.  
 Cylinder Barrel " " Sample Casting inspected.  
 Crankshaft " " ?  
 Piston Cover " " Dowler.  
 Chaincase " " Dowler...  
 Cam Housing Cover Pattern Made. Sample Casting inspected.  
 Rocker Box Cover. Pattern making here.  
 Contact Breaker Cover. " " "  
 Timing Chain Tensioner Pad. ?  
 Driving " " " ?

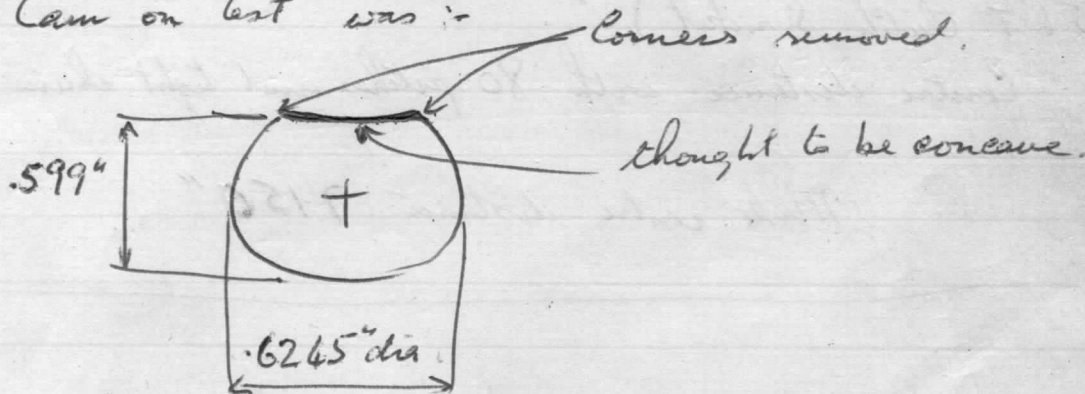
Notes on Arrangement re. the following:-

- Thrust Washers on O H V Rockers towards the front on both Rockers.
- Adjustment of Foot Change.
- Load at which release valves work.
- Value Timing.

125 cc Model "RE-2"

24-10-50

Alternator Cam on test was:-



Clutch Worm Return Spring "RE-2" Wire gauge = 22 gauge (.028")

Cylinder Head " " Mr Foster.  
 Cylinder Barrel " " Sample Casting inspected.  
 Crankshaft " ?  
 Piston Cover " " Dowler.  
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 Cam Housing Cover Pattern Made. Sample Casting inspected.  
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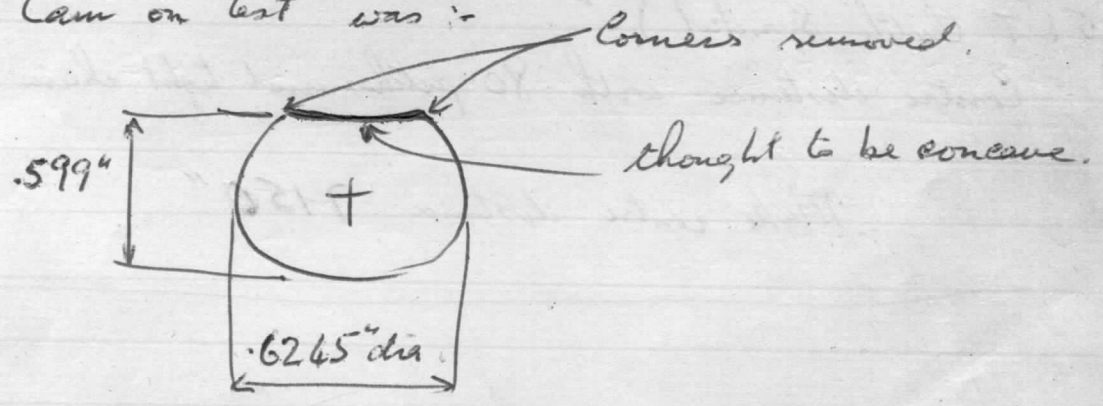
Notes on Arrangement re. the following:-

- Thrust Washers on O.V.V. Rockers towards the front on both Rockers.
- Adjustment of Foot Change.
- Load at which release valves work.
- Value Timing.

12.5 cc Model "RE-2"

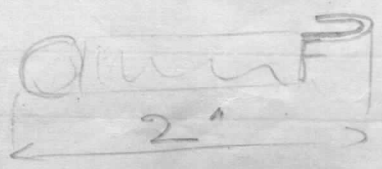
24-10-50

Alternator Cam on test was:-



Clutch Worm Return Spring "RE-2"  
2" overall length

Wire gauge = 22 swg (.028)  
Piano Wire  
O/Dia = .325"  
Double Eye 5/16" Int. Dia.  
Hook 1/16" internal 3/16" long



20-11-50

248 cc OHV. MODEL

Engine-Gearbox centres increased from  $6.876''$  horizontally to  $6.920''$   
 $6.874''$

As Gear-change Arm Centre is not moved horizontally with Gear-box centre but the Arm remains in the same position centrally between the two gear-shafts the effective length of the foot-change internal lever is reduced slightly and the gear-change arm could be lengthened to give a more central position of the pads in relation to the sliding gears.

500 cc. OHV. Single.

13-12-50

Cylinder Head (Single Port) 24808 (918A)

Faster Head than later Single Port.

1" distance piece reqd. on inlet flange to enable carburettor to connect with Air Cleaner.

J-28 Unit-construction

5-4-51

25 T Engine Sprocket }  $3\frac{1}{8}''$  Pitch.  
56 T Clutch Sprocket }

Centre distance with 80 pitches and tight chain =  $7.163''$

Make centre distance  $7.150''$

13-7-51

"R.E.-2"

CHECK ON ENGAGEMENT & DISENGAGEMENT OF K'START RATCHETS

C'case D/Side. From  $\text{\textcircled{C}}$  to inside Bearing = 1.7555"  
 1.7495"

C'case T/Side. From  $\text{\textcircled{C}}$  to inside Bearing = .989"  
 .983"

From  $\text{\textcircled{C}}$  to outside of fixed ratchet = 2.760"  
 2.735"

With mainshaft and clearance this becomes 2.782"  
 2.735"

From  $\text{\textcircled{C}}$  to inside of moving ratchet in engaged position = 2.707"  
 2.667"  
 with k'starter quadrant pushed outwards against inner face  
 of cover.

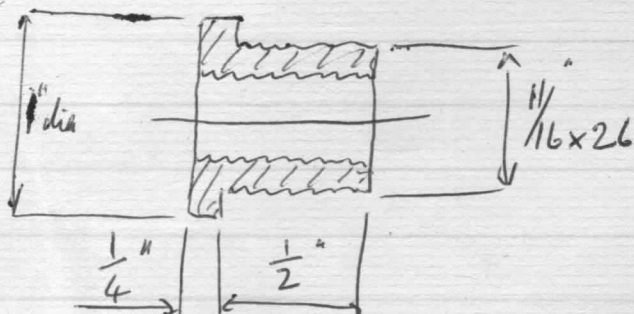
$\therefore$  minimum engagement =  $2.735 - 2.707 = .028$ "

and maximum movement for disengaging =  $2.782 - 2.667$ "  
 = .115" + end clearance of k'starter spindle.

28-8-51

500cc Twin Cylinder Heads on International Machine

Bush screwed in to take Sparking Plug.



& drilled & Tapped.

Head Spotfaul  $3/64$ " off-centre  
 to ~~miss~~ miss valve seating.

Matchless 500cc OHV TWIN 11-9-51

Outer Valve Spring.  $15/16$ " O.D.M x  $13/16$ " long in position.

Valves  $9/32$ " dia Stem?

$13/8$ " dia Inlet } Head dia of valves  
 $15/16$ " dia Exhaust }

12-9-51

248 cc OHV model.

Primary Drive. Chain No 110036  $\cdot 375$ " pitch &  $\cdot 25$ " Roller  
 $\times \cdot 225$ "  $\times$  70 pitches.

Engine Sprocket 23T.

Clutch Sprocket 49T.

Centre Distance without a tensioner =  $6.177$ "

Gear type oil pump.

24 D.P. 15T.  $\cdot 625$  pitch dia. & centres.

Spiral gear drive from camshaft to pump: Normal D.P. = 24. 14T.  
 $\cdot 825$ " pitch dia. & centres.

Worm Gear drive from mainshaft.

$\frac{1}{8}$ " Gear centres. 3 start worm. 14T wheel. 24 D.P.

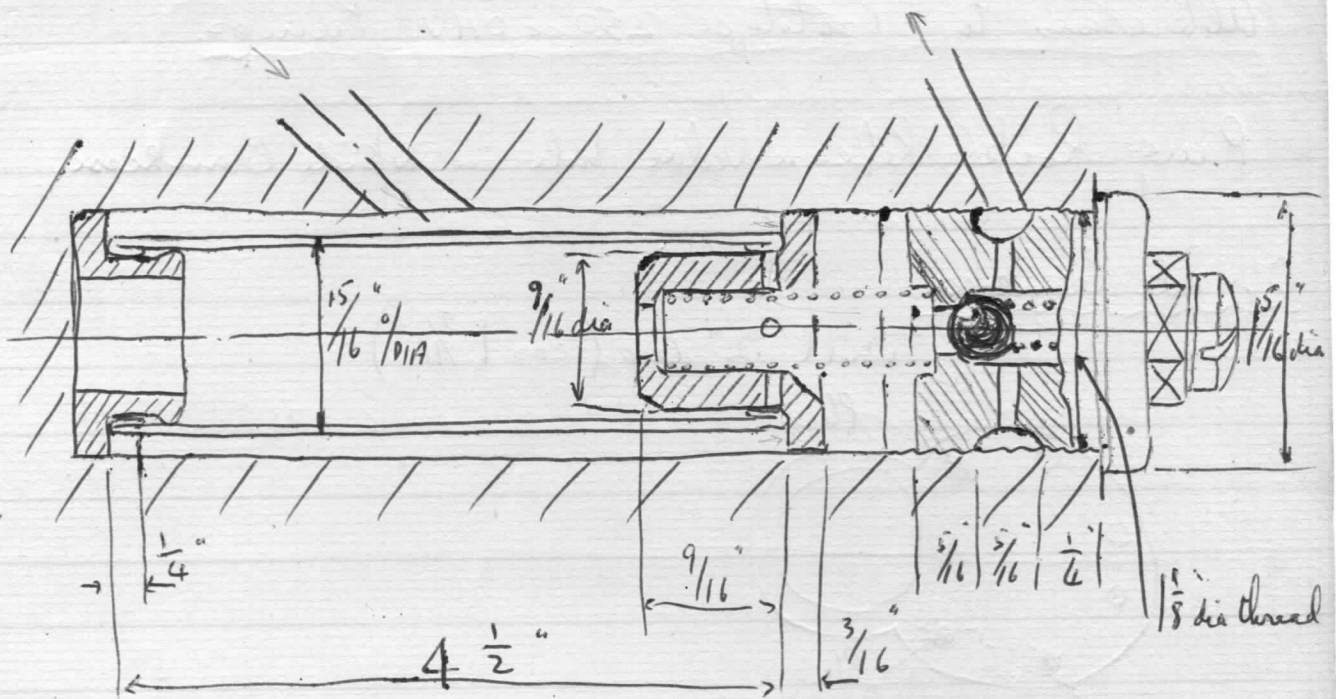
$2\frac{5}{8}$ " from joint face of crankcase.

Camshaft Drive

40 pitches 12T 24T  $\frac{3}{8}$  pitches -  $4.061$ " centres

40 pitches 13T 26T  $\frac{3}{8}$  pitch -  $3.762$ " centres

42 pitches 14T 28T  $\frac{3}{8}$  pitch -  $3.845$ " centres.



Matchless 500 twin felt oil cleaner

Rough Valve Timing on Prototype 250

I opens  $1/4$ " B.T.D.C. ( $30^\circ$ )

I closes 1" A.B.D.C. ( $77^\circ$ )

Ex Opens  $5/8$ " B.B.D.C. ( $60^\circ$ )

Ex Closes  $3/8$ " A.T.P.C. ( $38^\circ$ )

Advance  $8\frac{1}{2}^\circ$  on engine shaft.

Valve Timing on Prototype 250 with revised cams.

I opens  $13/64$ " B.T.D.C. ( $26\frac{1}{2}^\circ$ )

I closes  $7/16$ " A.B.D.C. ( $51^\circ$ )

Ex opens  $25/32$ " B.B.D.C. ( $68\frac{1}{2}^\circ$ )

Ex closes  $25/64$ " A.T.D.C. ( $37^\circ$ )

In correct

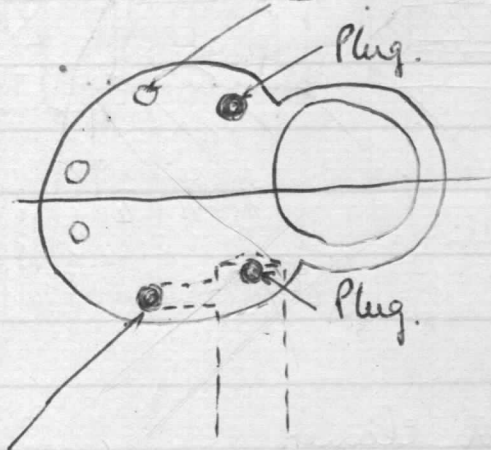
Ex  $5^\circ$  late

Alterations to Prototype 250cc OHV Pump. ✓

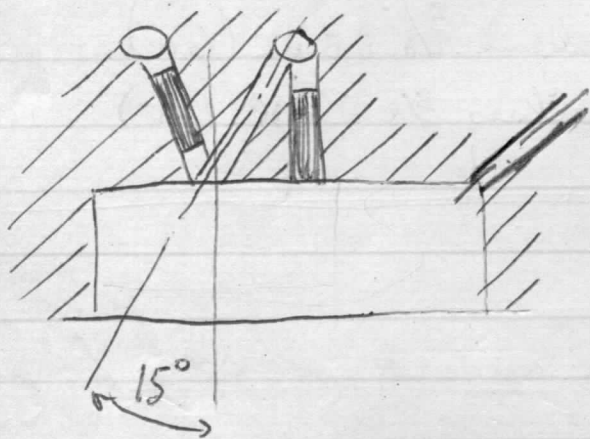
Plug Rocker Release Valve hole in N/Side Crankcase.

Off Side Crankcase.

Drill 2" deep (was  $1\frac{5}{16}$ " )



Plug & Redrill  $1\frac{5}{16}$ " deep at  $15^\circ$  to break into hole from Pump.



Vent back half of Pump into crankcase.

Oil Return hole to tank - Re-drill  $\frac{1}{2}$ " dia & Tap  $\frac{5}{16}$ " BSF x  $\frac{1}{2}$ " deep.

Fill in G-roove on Pump Disc face.

Open out top plunger hole to  $\frac{.251}{.250}$ " dia to take plunger W.13906 (Std G-2 feed)

Remove two extra pieces on Disc.

Re-lap in disc and assemble without extra washers and cap.

248cc OHV MODEL

9-4-52

Primary Chain 110 036 x 72 pitches  
 23T - 52T Centre distance = 6.224" } 6-2 Top Gear.

Secondary Chain 110 048 x pitches  
 16T - 44T

Camshaft Chain 114 500 x 54 pitches  
 19T - 38T Centre Distance = 3.900"

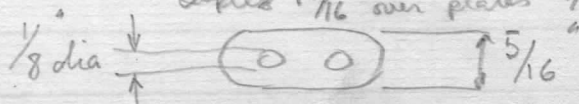
114 500 x 56 pitches = 4.225" Centre Distance.  
 114 500 x 58 pitches = 4.548" " "  
 114 500 x 50 pitches = 3.246" " "  
 114 500 x 52 pitches = 3.574" " "  
 14T-28T 114 500 x 38 pitches = 2.582" " "  
 " " " x 40 pitches = 2.908" " "

7/32

8 m/p Chain (duplex)  $\frac{15}{32}$ " over plates  $\frac{17}{32}$ " over rivets +  $\frac{1}{32}$ " side movement



$\frac{3}{8}$ " pitch Chain. Simple  $\frac{13}{32}$ " over plates  $\frac{1}{2}$ " over rivets +  $\frac{1}{32}$ " side movement.  
 Duplex  $\frac{13}{16}$ " over plates  $\frac{7}{8}$ " over rivets +  $\frac{1}{32}$ " side movement.



Clutch

Narrower ~~engine~~ sprocket. to allow for movement of clutch



Cylinder Base Washer - Manilla "A" Paper .010" thick.

Push Rod Tube Washer Hallite

Inlet Valve & Exhaust Valve EN54 Head 1 1/2" dia. 1/16" dome. 1/32" parallel to chamfer at 45°. 3/4" rad to .341" +0/-1/2 stem dia. 18" incl taper for split collar.

Value Spring (Outer) 1 1/8" inside dia +.04/-0. 1 3/4" long. 85-90 lbs compressed to 1 3/32" 5 1/2 coils

" (Inner) 1 3/16" inside dia. 1 3/4" long 35-40 lbs " " 7/8" 6 1/2 coils point to point. Round & grind ends square.

Value Collar Top Hecla C8. Split Collar M.S. Stem Cap K.E. Cone Steel Harden & Temper.

Value Guide MK. 3. Nickel Chrome Alloy Iron Rod. Quench in oil at 875°C. for C.I. Heads. Harden & G.

.3447" Bore .624" DIA.  
.3437" Bore .623" DIA.

Value Spring Collar (Bottom) Ann M.S.

Value Rockers. 2% Nickel steel stamp. C.H. & G.

Rock Bearing & Caps C.I.

Push Rods "A" Qual. Tube. Hard Drawn. 3/8" DIA x 19 SW.G.

Push Rod End (Top) K.E. Cone Steel. Harden & Temper.

" (Bottom) M.S.

" Caps U.B.A.S. C.H. Cup.

Tappets. U.B.A.S. C.H. & G. .375" dia stem. 1" dia x .104" lead .374" .094" lead  
5/32" Rad Spher. End. with 23° angle per side & 1/16" rad.

Tappet Guides MK. 3. Nickel Chrome Alloy Iron Rod. ground to (Inlet & Short) 1" dia +.005" Quench in oil at 875°C.

(Exhaust-Long) .875" .376" dia bore. 2 1/8" & 2 1/2" overall lengths  
.874" DIA .375" dia bore.

Cams U.B.A.S. C.H. & G. 1" dia base circle. 5/16" lift. .750" bore.

Gears - 2 pitch dia. 40T. 20 D.P. .749" bore.

Cam Bushes Carobronze Tube .760" Dia x .616" Bore  
.764" Dia x .620" Bore  
Bore? DIA?

Cam Spindles Phoenix Steel "CH & G" .6245" dia for Cams.  
.6235" dia for Cams.

Dia of holes in case = .624" .626"  
for cam spindles. .623" dia. .625" dia into crankcase.

holes for idler pins in spindles = .498" dia. Tappet Guides .875" dia  
.497" dia. .874" dia

hole for filler collar = 1/4" dia x 20 T.P.I. R.H. Whit.

248 cc OHV Model

SHORT STROKE

70<sup>mm</sup> Bore x 64.5<sup>mm</sup> Stroke

Camshaft Drive

19 T — 38 T

114500 x 52 pitches.

Centre Distance = 3.574"

Combustion Chamber

1 1/2" rad

905" (23mm) high.

Primary Drive

3/8" pitch

21 T — 52 T

70 pitches.

Centre Distance = 5.991"

21 T — 50 T 70 pitches = 6.224" e/distance

MODEL J-S. ROLLER BEARING BIG-END.

Crank Rod. .693" 1-9325"  
-688" wide, 1-9315" dia bore.

Flywheels. .9995"  
.9990" dia for crankpin. Chamfer  $\frac{1}{16}$ " x 45°  
-703" wide 1.000" from mainshaft face.  
.997"

.121" instead of .106" dimension

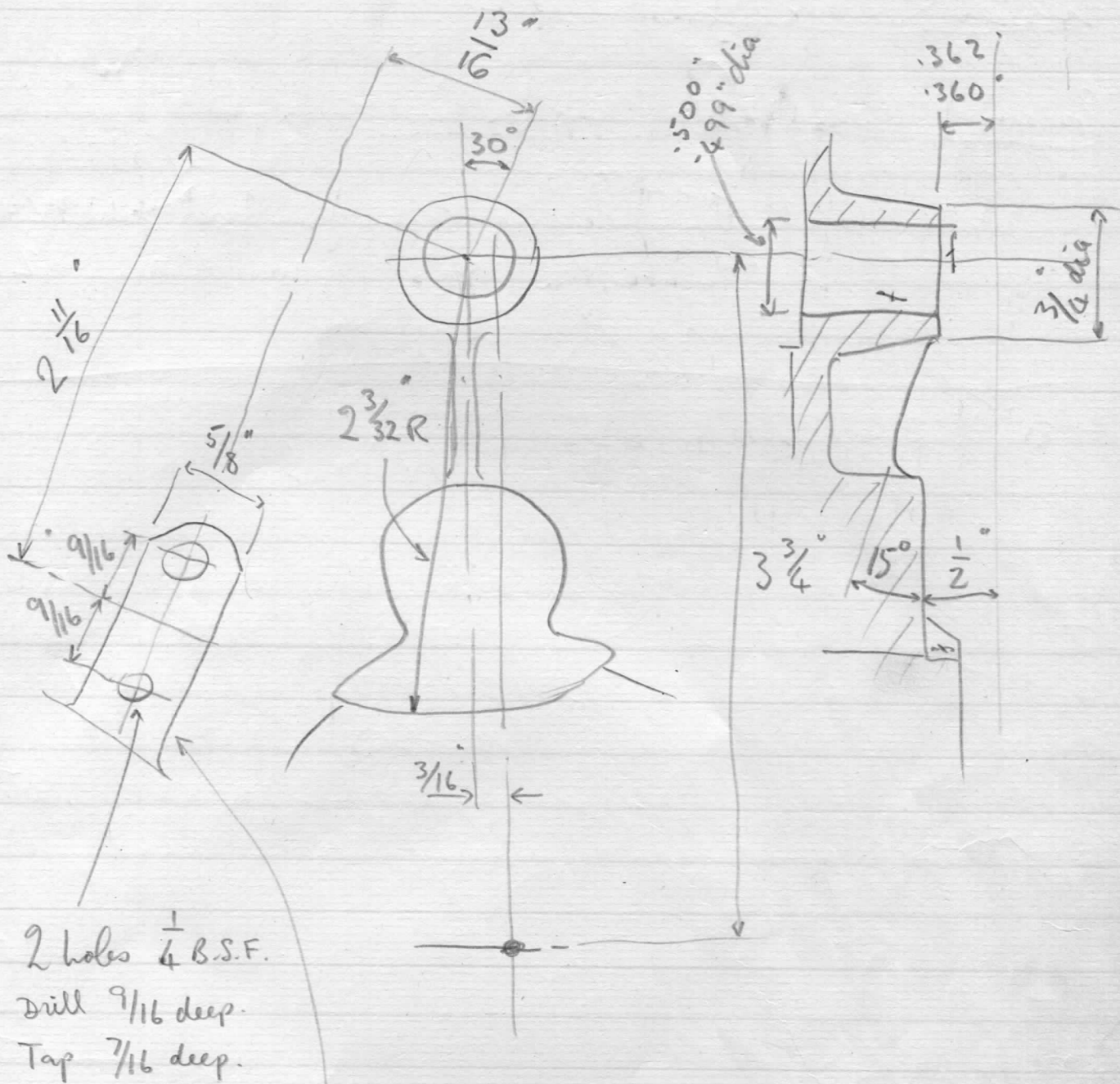
Peg hole N°43 drill (.089")  $\frac{5}{16}$ " deep  $\frac{7}{8}$ " from center.

Recess for Thrust Washer 2.132" dia.  
2.128"

Oil hole in T/Side 22° and  $\frac{9}{32}$ " back from  
big end thrust washer face.

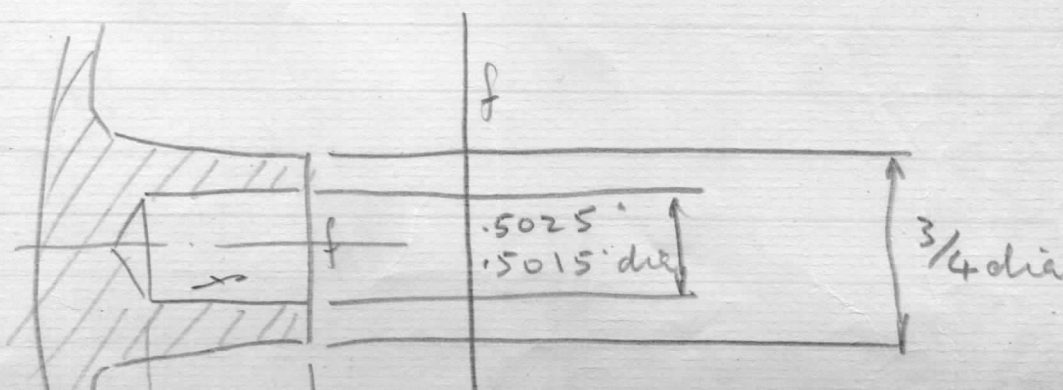
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Modification to Crankcase 36503



This face  $\frac{1}{4}''$  back from front face.

Section thro' boss on Timing Cover 37449



✓  
Albion Gearbox (Excelsior Type)

Position of Lunde Operator Anchor Pin

Distance of Bottom of bearing Recess from front face = 4.922"  
" " Anchor Pin Boss " " " = 2.057"

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$$\frac{49}{23} \times \frac{49}{18}$$

248 cc OHV MODEL

SHORT STROKE 70<sup>M</sup>/M BORE x 64.5<sup>M</sup>/M STROKE

Primary Drive 23T - 49T Centre Distance = 6.178"  
Chain N° 110038 x 70 pitches.

Camschaft Drive 14T - 28T Centre Distance = 2.582"  
Chain N° 114500 x 38 pitches.

Angle of valves = 65° included.

Final Drive 16T - 49T Chain N° 110046  
= Top Gear Ratio 6.5 with 3.25 x 18 tyre on WM3-18 rim

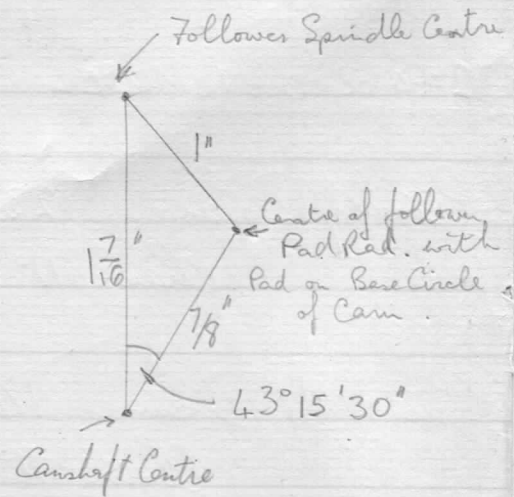
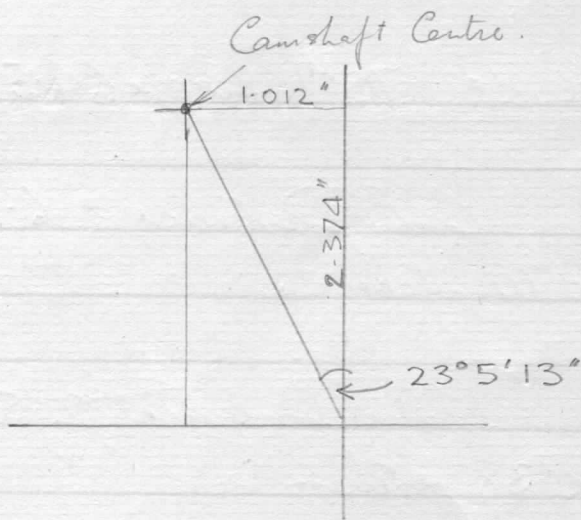
Final Drive for 3.25 x 17" tyre = 18T - 49T  
Chain N° 110046  
= Top Gear Ratios 5.8 3rd 7.83 2nd 10.64 1st 16.965

Gear drive to pump 20 D.P.

24T — 1.200° Pitch dia 1.300° o/dia 2 OFF

23T — 1.150° Pitch dia 1.250° o/dia 1 OFF (IDLER)

Generator (motor) max Taper 15/16 dia 7/8 long.

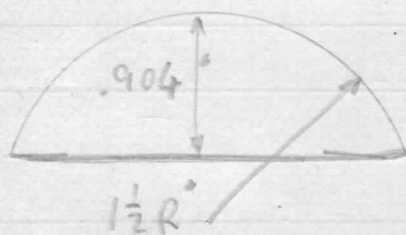


### Kickstarter Ratios

	<u>Primary Drive</u>	<u>K'starter</u>	<u>K'starter/Engine Ratio</u>
Meteor	32:56 - 1.75	18:30 - 1.67	$1.67 \times 1.75 = 2.92$
J	21:42 - 2	18:30 - 1.67	$1.67 \times 2 = 3.34$
G	19:42 - 2.2	18:30 - 1.67	$1.67 \times 2.2 = 3.67$
J-S&500TWIN	25:56 - 2.24	18:30 - 1.67	$1.67 \times 2.24 = 3.74$
New 250	23:49 - 2.13	15:27 - 1.8	$1.8 \times 2.13 = 3.83$
S	17:42 - 2.47	15:27 - 1.8	$1.8 \times 2.47 = 4.45$

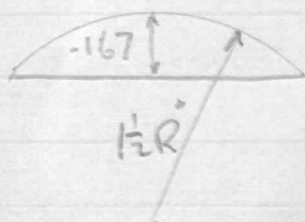
# Compression Ratio

9-3-53



$$V = 3.1416 \times 0.904^2 \times \left(1.5 - \frac{0.904}{3}\right)$$

Cylinder Head = 3.0788 cu ins.



$$V = 3.1416 \times 0.167^2 \times \left(1.5 - \frac{0.167}{3}\right) = 0.1257 \text{ cu ins.}$$

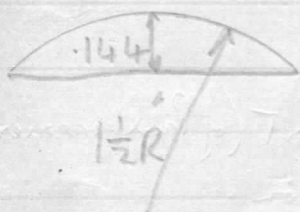
$1 \frac{13}{32}$  dia  $\times$   $\frac{1}{32}$  deep.

$$V = 1.553 \times 0.031 = 0.0481 \text{ cu ins.}$$

$1 \frac{7}{16}$  dia  $\times$   $\frac{1}{32}$  deep.

$$V = 1.623 \times 0.031 = 0.0503 \text{ cu ins.}$$

Inlet Valve = 0.2241 cu ins. -



$$V = 3.1416 \times 0.144^2 \times \left(1.5 - \frac{0.144}{3}\right) = 0.0942 \text{ cu ins.}$$

$$1 \frac{5}{16} \text{ dia} \times \frac{1}{32} \text{ deep} = 1.353 \times 0.031 = 0.0419 \text{ cu ins.}$$

$$1 \frac{1}{32} \text{ dia} \times \frac{1}{32} \text{ deep} = 1.418 \times 0.031 = 0.0440 \text{ cu ins.}$$

Exhaust Valve = 0.1801 cu ins. -

Estimated Volume of Recesses in Valve Heads = 0.081 cu ins. +

3.0788

0.2241

3.1598

0.0810

0.1801

0.4042

3.1598

0.4042

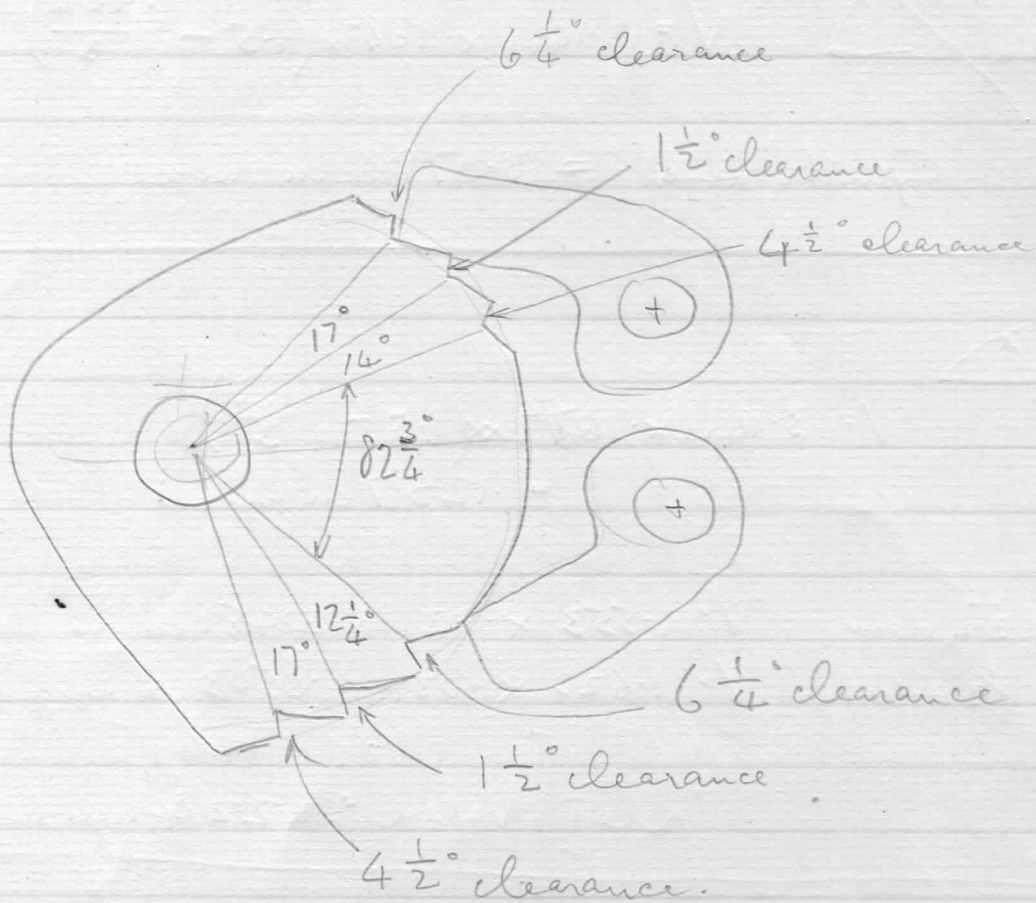
3.7556

cu ins = 45.15 cu cms  
Volume of Combustion Chamber

$\frac{9}{64}$  dome on piston (Std "Meteor") = 0.38 cu ins = 6 cu cms.

248cc Frot Change

5-6-53



$17^\circ + 1\frac{1}{2}^\circ$  working clearance =  $18\frac{1}{2}^\circ$  total movement (of Paul carrier) in each direction.

EN. 41 Nitriding Steel suitable for cam followers

## J-S Compression Ratios.

Volume of Combustion Chamber with Std Piston & Gasket calculated to be 90.73 cc. (Taking Std CR as 6.5:1)

Difference in volume though not using gasket (allowing .04" thickness) = 3.447 cu ins

Volume of Std Piston "dome" = 1.643 cu ins

Volume of Combustion Chamber with flat top piston & without gasket would be  $90.73 \text{ cc} + 1.643 \text{ cu ins} - 3.447 \text{ cu ins}$   
 $= 90.73 \text{ cc} + 1.298 \text{ cu ins}$   
 $= 90.73 \text{ cc} + 21.27 \text{ cc} = 112 \text{ cc.}$

Volume required to give 9.5:1 C.R. = 58.7 cc.

Volume of piston dome to give this ratio =  $112 - 58.7$   
 $= 53.3 \text{ cc} = 3.25 \text{ cu ins.}$

Allowing .05 cu ins for valve "cut-aways" then "dome" should equal 3.3 cu ins

Height of dome =  $23/32$ "

C.R. with Std Piston & without gasket found to be 7.1:1

Volume of Combustion Chamber = 81.8 cc. = 4.99 cu ins

Volume of Std Piston "Dome" = 1.643 cu ins.

Volume of Combustion Chamber with flat top piston & without gasket would be  $4.99 + 1.643 = 6.633 \text{ cu ins.}$

Volume required to give 9.5:1 CR = 58.7 cc = 3.581 cu ins.

Volume of piston dome to give this ratio =  $6.633 - 3.581 = 3.052 \text{ cu ins.}$

Allowing .15 cu ins for valve "cutaways" then "dome" should equal 3.2 cu ins

Height of dome =  $45/64$ " (Rad =  $2\frac{1}{4}$ " )

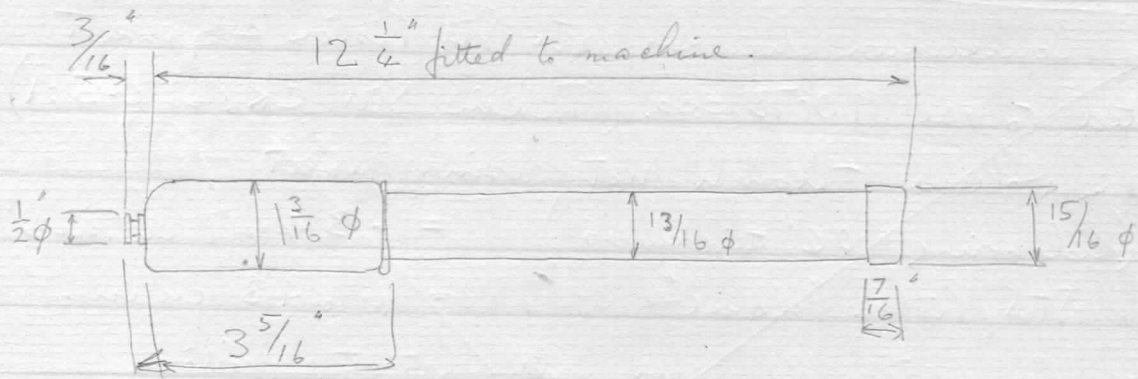
Volume required to give 8:1 CR = 71.3 cc = 4.349 cu ins

Volume of piston dome to give this ratio =  $6.633 - 4.349 = 2.284 \text{ cu ins.}$

Allowing .07 cu ins for valve "cutaways"

then "dome" should equal 2.35 cu ins.

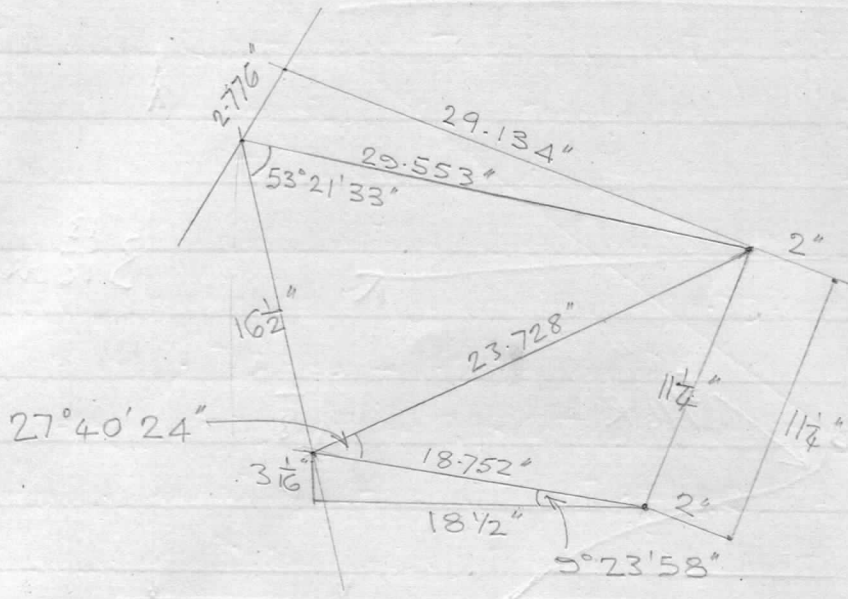
Height of dome =  $17/64$ "



Tyre INFLATOR 21/9/53

248cc OHV MODEL FRAME

7-9-53



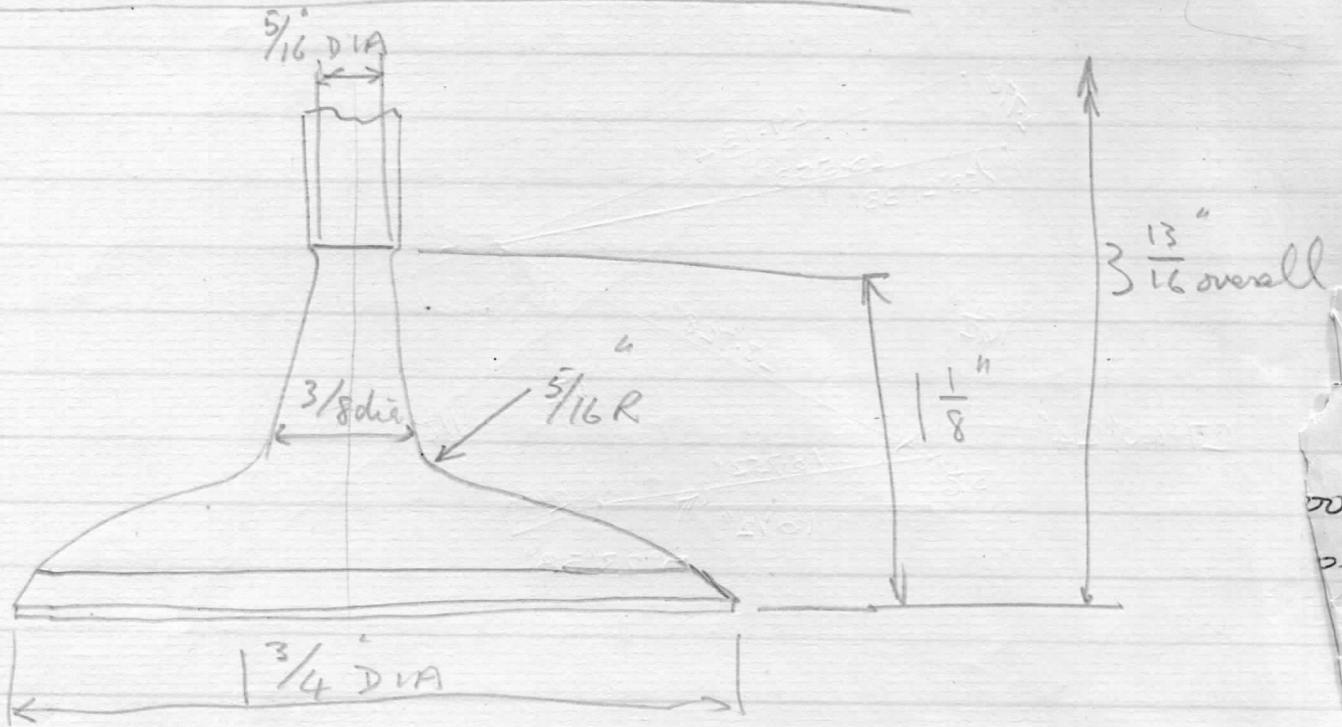
250cc machine 9-10-53

O/dia of Rear wheel 17" Rim with 3.25 tyre =  $24\frac{3}{4}$ "  
O/dia of Front " 17" " " ? " =  $24\frac{1}{4}$ "

.256  
257  
359



500 Gold Star BSA In Valve Head



TO IN  
D-256